

Mathematical Modeling and Performance Evaluation of Electro-Hydraulic Servo Actuators



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OUTLINE

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- 3. Challenges & Solutions With MATLAB/Simulink
- 4. Physical and Equation Based Models
- 5. Modelling Nonlinearities
- 6. Translating Complex Systems into Mathematical Models
- 7. Performance Parameters Experimental vs Simulation Results
- 8. Model Linearization & Order Reduction for HILS Implementation
- 9. Loading Rig and Robust Force control Design
- 10.Design Optimization for Software Thresholds

11.Conclusions









- ✤ Unstable Aircraft
- State of art high performance Actuators
- Optimized Envelope and Weight
- Build in Redundancy
- Reliable development tools
- Design of Test Systems
- Independent validation and verification
- Stringent Certification Process - RCMA
- Airworthy /MIL Qualification Requirements











- Actuators Connected to Flight Control Computer which houses the Servo Electronics or the Drive Circuitry for various units namely LVDTs, motor, solenoid valves etc.
- The flight control computer contain Software Algorithms for Health Monitoring (Built in Tests, Inhibition etc.) and implementation of Autopilot and Control Law functionalities.
- Power to overcome Aerodynamic Load provided by Hydraulic Power Source







Modelling Requirements in Servo Actuators



Key Requirements

- Need for reliable modelling and simulation tools for state of art Direct Drive Valve (DDV) and Electro Hydro Servo Valve (EHSV) based Hydraulic Actuators
- Robust Controller Design and Stability Analysis
- Modelling Non Linearities like Friction, Hysteresis, Backlash and Dead Band which are very important in characterizing low amplitude behavior
- Real time and Hardware/Software Implementation (HIL and SIL simulations)
 Why MATLAB/SIMULINK is every Flight Control System Designer's Choice
- Ease of modelling with functional blocks and detailed components
- Vast set of libraries make it very easy to model and simulate inter disciplinary system like Fly By Wire Flight Control systems
- Physical system models and equation/signal based model can be easily coupled
- > Easy interface with Aircraft model, other software and large data handling
- > Quick Simulation of different test conditions and actual environment









Objective

- 1. Non Linear Mathematical Model of a Electro-Hydraulic Servo Actuator
- 2. Controller Design to meet Dynamic Requirement
- 3. Actuator Test Rig for Loaded Performance of the Actuator
- 4. Robust Force Control Design for the Rig

Key Learnings

- Fly by wire Flight Control Systems and Actuators
- Equation and Physical Component Based Model
- Modelling Hydraulic Components
- ✤ Servo Valve Dynamics –EHSV
- Model Reduction Techniques & Linearization
- Modelling Non –Linearities
- Controller Design Optimization & DOE
- Loading Rig Simulation & Force Controller Design
- Software logic verification using model









Modelling Hysteresis & other Non Linearities

Pressure Fluctuations due to Fluid Inertia and Capacitance

Non Linear Model Frequency Response

Analysis at Different Test Points

Optimal Force Control

Functional Blocks/Non Linear Blocks in Simulink

Resistive Pipe Elements in Sim Hydraulics

Customized MATLAB Code for FRA

MATLAB Code generation for Automation

Robust Control Toolbox in Simulink





Physical Component Based Model of an EHSV Based Actuator



Control Electronics is modelled with Transfer Functions









- ✤ Easy to Implement Certain Features
- Can be easily Coupled with Physical Component Based Model
- Nozzle Flapper Equations

$$Q_{1} - Q_{3} + A_{s} \frac{dx}{dt} = \left(\frac{V_{0} - A_{s}x}{\beta}\right) \frac{dp_{1}}{dt} \qquad Q_{2} - Q_{4} - A_{s} \frac{dx}{dt} = \left(\frac{V_{0} + A_{s}x}{\beta}\right) \frac{dp_{2}}{dt} \qquad Q_{3} + Q_{4} - Q_{5} = \frac{V_{3}}{\beta} \frac{dp_{3}}{dt}$$
$$F_{1} - F_{2} = \left(P_{1} - P_{2}\right) * An + \frac{\pi C df^{2}}{4} \left[P_{1}(Xfo - Xf)^{2} - P_{2}(Xfo + Xf)^{2}\right]$$

Simulink Implementation







Input - Output with Backlash





Translating Complex Systems into Models



Clearance and seal friction between moving THS2 THS1 THS3 THS4 Self Centering Actuator * with cap and body R., 16 6 . . . - r **... Complex Dynamics** TF1 %_**⊷** TF2 MTR2 JE3 TF4 pistonMass - RWe RWC ✤ Failure Logic Simulation TS3 TS2 capMassLeft Centring THMC_A capMassRight spring Clearance and seal friction between piston and moving cap Moving cap CCF on the right • pistonOutput DAHC centringOrificeA BVA BVB centringOrificeB • ₩Ò₳⊷⊷₽≍<u></u>≁י 2 flowFromValvePortB IowFromValvePortA 3 supply Pressure ✤ Dual Tandem Actuator with Figure 30. CSAS Simulink Model. (Including Bypass Valve) Force Addition (2)A1 (3)B2 11A2 (4)B1 ✤ Hydraulic Failure Simulation during external Force Cylinder Friction1 B --5 R Cylinder Friction Hard Convert 2 Convert 3 Hard Stop2 Convert1 ñ Convert4 B2 A1 A2 B, C 6 e i∢ Motion Chamber2 Chamber3 Chamber4 Chamber1 Sensor



Frequency Response for Nonlinear Model vs Experimental Data



Important Performance Requirements for Servo Actuators

- Stall Load, No Load Rate, Frequency Response, Dynamic Stiffness
- ✤ Failure Transients

R&D ORGA









Why Modelling Nonlinearities is important?





No load Rate-Experimental data Match















Nichols Plot Showing Linear Behavior of the Servo Actuator







Actuator Performance under Load

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Key Features

- Flight Control Actuator in Position Loop
- Loading Actuator in Force Control Loop
- Force correction due to angle change
- Force control within ± 10 % of commanded load for all conditions

Design Challenges

- ✤ Modeling dynamic Characteristics
- Set point is changing with time
- Cater for various strokes, loads and frequencies
- Disturbance in the force caused by movement of flight control actuator







- Optimum sizing of the servo valve- rated flow, pressure gain, dead band, response
- Optimum size of loading cylinder, pump and other components
- Accumulative pressure drop and power consumption at all operating conditions



Loading Performance & Stability Analysis Results



High Load, Low Frequency

E R&D ORGAN

1:2000 CE



Very Low Load, High Frequency











. . . .

10¹

Frequency (Hz)

Indigenous Actuators

-200

10⁻¹

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10⁰

Powering the wings

 10^{2}





Software Thresholds & parameters

ALD ORG









- □ Modelling and Simulation as an effective Design Tool
- □ MATLAB/Simulink are part of design process
- Modelling aspects and simulation results are of paramount importance for all design reviews related to actuators/test rigs
- Quick Verification of Preliminary Design with Model
- Estimation of System Parameters, which can't be measured easily
- Controller design for non linear model to meet dynamic requirement
- Excellent match with experimental data for all performance parameters
- Reduced order model for HILS C Code

For The Loading Rig

- Prediction of actuator performance under load
- Prediction of Hydro Mechanical Resonant Frequencies
- Additional feed forward force control loop with adaptive gains based on dynamics of the disturbance caused by the movement of flight control actuator is essential for robust force control design of loading actuator.
- The design criteria to keep the force variation within ±10% of commanded force and keep the phase lag within ±10 degrees in the force control loop is achieved.







Thank YouAny Questions ??



